CLAIMS

1. Crank shear, especially for cutting rolled strip (22), which comprises two pairs of blades (3, 4) that can be mounted on blade holders (1, 2), wherein the blade holders (1, 2) are supported opposite each other in a vertical plane (x-x) in a pair of eccentric drive shafts (5, 6) and are pivoted on torque supporting levers (7, 8) in double-joint mechanisms (9, 10) in interaction with two hydraulic control units (11, 12) that act on these double-joint mechanisms (9, 10), wherein the blade holders (1, 2) form axially parallel pairs of bearing surfaces (16-19) for the pairs of blades (3, 4) on approximately radial projections (13-15), with the upper pair of blades (3) arranged on inner, oppositely oriented bearing surfaces (16, 17) of a curved recess (20) of the upper blade holder (1), and with the lower pair of blades (4) arranged on the outer bearing surfaces (19, 18) of a relatively narrow projection (15) oriented towards the recess (20), and wherein in a spread position of the torque supporting levers (7, 8) of approximately 90° and at the greatest separation (d) of the eccentric shafts (5, 6) and a running direction (21) of the rolled strip (22) towards the supporting levers (7, 8), a passage position of the crank shear is reached, in which the control unit (12) on the lower supporting lever (8) of the double-joint mechanism (10) is fully retracted, and the control unit (11) on the upper supporting lever (7) of the double-joint mechanism (9) is fully extended (Figure 2).

2. Crank shear, especially for cutting rolled strip (22), which comprises two pairs of blades (3, 4) that can be mounted on blade holders (1, 2), wherein the blade holders (1, 2) are supported opposite each other in a vertical plane (x-x) in a pair of eccentric drive shafts (5, 6) and are pivoted on torque supporting levers (7, 8) in double-joint mechanisms (9, 10) in interaction with two hydraulic control units (11, 12) that act on these double-joint mechanisms (9, 10), wherein the blade holders (1, 2) form axially parallel pairs of bearing surfaces (16-19) for the pairs of blades (3, 4) on approximately radial projections (13-15), with the upper pair of blades (3) arranged on inner, oppositely oriented bearing surfaces (16, 17) of a curved recess (20) of the upper blade holder (1), and with the lower pair of blades (4) arranged on the outer bearing surfaces (19, 18) of a relatively narrow projection (15) oriented towards the recess (20), and wherein in a position of the upper supporting lever (7) that is downwardly inclined towards the rolled strip (22) with the upper hydraulic control unit (11) of the double-joint mechanism (9) retracted, and in a position of the lower supporting lever (8) that is upwardly inclined towards the rolled strip (22) with the lower control unit (12) of the double-joint mechanism (10) fully extended, and with the greatest separation (d) of the eccentric shafts (5, 6), the passage position through the shear is reached (Figure 4).

3. Crank shear in accordance with Claim 1 or Claim 2, characterized by the fact that in a spread position of the torque supporting levers (7, 8) of approximately 90° and at the shortest separation (D) of the eccentric shafts (5, 6) and a running direction (21) of the rolled strip (22) towards the supporting levers (7, 8), a position of the pair of blades (3) for the cropping cut at the leading end (23) of the strip is reached, in which the hydraulic control unit (11) on the upper supporting lever (7) of the double-joint mechanism (9) is fully extended, and the hydraulic control unit (12) on the lower supporting lever (8) of the double-joint mechanism (10) is fully retracted (Figure 1).

4. Crank shear in accordance with Claim 1 or Claim 2, characterized by the fact that in an approximately horizontal parallel position of the torque supporting levers (7, 8) opposite the running direction (21) of the rolled strip (22) and at the shortest separation (D) of the eccentric shafts (5, 6), and with the upper double-joint mechanism (9) and lower double-joint mechanism (10) extended approximately linearly, a position of the rear pair of blades for cutting the tail end (24) of the strip is reached, in which the hydraulic control unit (11) on the upper supporting lever (7) of the double-joint mechanism (9) is fully retracted, and the control unit (12) on the lower supporting lever (8) of the double-joint mechanism (10) is fully extended (Figure 3).